

MAE/CE 671 Continuum Mechanics

Text

1. Show that the Jacobian is equal to the square root of the determinant of the metric tensor.

Let \mathbf{F} be the displacement gradient:

$$\mathbf{F} = \begin{bmatrix} \frac{\partial z_1}{\partial x_1} & \frac{\partial z_1}{\partial x_2} & \frac{\partial z_1}{\partial x_3} \\ \frac{\partial z_2}{\partial x_1} & \frac{\partial z_2}{\partial x_2} & \frac{\partial z_2}{\partial x_3} \\ \frac{\partial z_3}{\partial x_1} & \frac{\partial z_3}{\partial x_2} & \frac{\partial z_3}{\partial x_3} \end{bmatrix}$$

$$G_{ij} = \mathbf{G}_i \cdot \mathbf{G}_j = \frac{\partial \mathbf{R}}{\partial x_i} \cdot \frac{\partial \mathbf{R}}{\partial x_j} = \frac{z_i \mathbf{i}_i}{\partial x_i} \cdot \frac{z_j \mathbf{i}_j}{\partial x_j} = \frac{\partial z_i}{\partial x_i} \frac{\partial z_j}{\partial x_j}$$

$$G_{ij} = F_{mi} F_{mj}$$

$$\mathbf{G} = \mathbf{F}^T \mathbf{F}$$

$$\det[\mathbf{G}] = \det[\mathbf{F}^T \mathbf{F}] = \det[\mathbf{F}] \det[\mathbf{F}] = J^2$$

$$J = \sqrt{\det[\mathbf{G}]}$$

2. Let the motion of a fluid in steady-state flow be given in the Eulerian coordinates by

$$\begin{aligned}V_1 &= k z_1 \\V_2 &= -k z_2 \\V_3 &= 0\end{aligned}$$

a) Show that the motions and velocities in Lagrangian coordinates are

$$\begin{aligned}z_1 &= x_1 \exp[k(t - t_0)] & v_1 &= kx_1 \exp[k(t - t_0)] \\z_2 &= x_2 \exp[-k(t - t_0)] & v_2 &= -kx_2 \exp[-k(t - t_0)] \\z_3 &= x_3 & v_3 &= 0\end{aligned}$$

Pfaffian Form

$$\mathbf{R} = z_i \mathbf{i}_i$$

$$\frac{dz_1}{V_1(\mathbf{R}, t)} = \frac{dz_2}{V_2(\mathbf{R}, t)} = \frac{dz_3}{V_3(\mathbf{R}, t)} = dt$$

$$V_1 = k z_1$$

$$\int_{z_1(t_0)}^{z_1(t)} \frac{dz_1}{k z_1} = \int_{t_0}^t dt$$

$$\frac{dz_1}{dt} = k z_1$$

$$\frac{1}{k} \{ \ln[z_1(t)] - \ln[z_1(t_0)] \} = t - t_0$$

$$\frac{dz_1}{dt} - k z_1 = 0$$

$$\ln[z_1(t)/z_1(t_0)] = k(t - t_0)$$

$$sZ_1(s) - z_1(0) - kZ_1(s) = 0$$

$$z_1(t)/z_1(t_0) = \exp[k(t - t_0)]$$

$$(s - k)Z_1(s) - z_1(0) = 0$$

$$z_1(t) = z_1(t_0) \exp[k(t - t_0)]$$

$$Z_1(s) = \frac{z_1(0)}{(s - k)}$$

$$z_1(t_0) = x_1$$

$$z_1(t) = x_1 \exp[k(t - t_0)]$$

$$v_1(t) = k x_1 \exp[k(t - t_0)]$$

$$z_1(t) = z_1(0) \exp[k(t - t_0)]$$

$$z_1(t_0) = x_1$$

$$z_1(t) = x_1 \exp[k(t - t_0)]$$

alternate

$$v_1 = \frac{dz_1}{dt} = kz_1$$

Let

$$z_1 = A \exp(-\omega t)$$

$$\frac{dz_1}{dt} = -\omega A \exp(-\omega t)$$

$$-\omega A \exp(-\omega t) = k A \exp(-\omega t)$$

$$\omega = -k$$

$$z_1 = A \exp(kt)$$

$$z_1(0) = x_1$$

$$z_1 = x_1 \exp(k(t - t_0))$$

$$V_2 = -k z_2$$

$$-\int_{z_2(t_0)}^{z_2(t)} \frac{dz_2}{k z_2} = \int_{t_0}^t dt$$

$$-\frac{1}{k} \{\ln[z_2(t)] - \ln[z_2(t_0)]\} = t - t_0$$

$$\ln[z_2(t)/z_2(t_0)] = -k(t - t_0)$$

$$z_2(t)/z_2(t_0) = \exp[-k(t - t_0)]$$

$$z_2(t) = z_2(t_0) \exp[-k(t - t_0)]$$

$$z_2(t_0) = x_2$$

$$z_2(t) = x_2 \exp[-k(t - t_0)]$$

$$v_2(t) = -k x_2 \exp[-k(t - t_0)]$$

$$dz_3 = V_3(\mathbf{R}, t) dt$$

$$V_3 = 0$$

$$dz_3 = 0$$

$$z_3(t) - z_3(t_0) = 0$$

$$z_3(t) = z_3(t_0) = x_3$$

b) Transform the Lagrangian velocities into Eulerian coordinates to prove that the results in part (a) are recovered.

$$z_1 = x_1 \exp[k(t - t_0)]$$

$$x_1 = z_1 / \exp[k(t - t_0)]$$

$$z_2 = x_2 \exp[-k(t - t_0)]$$

$$x_2 = z_2 / \exp[-k(t - t_0)]$$

$$z_3 = x_3$$

$$x_3 = z_3$$

$$v_1 = kx_1 \exp[k(t - t_0)]$$

$$V_1 = kz_1 \exp[k(t - t_0)] / \exp[k(t - t_0)] = kz_1$$

$$v_2 = -kx_2 \exp[-k(t - t_0)]$$

$$V_2 = -kz_2 \exp[-k(t - t_0)] / \exp[-k(t - t_0)] = -kz_2$$

$$v_3 = 0$$

$$V_3 = 0$$

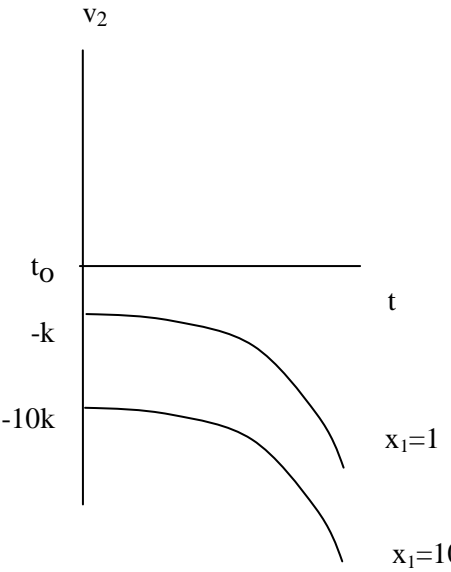
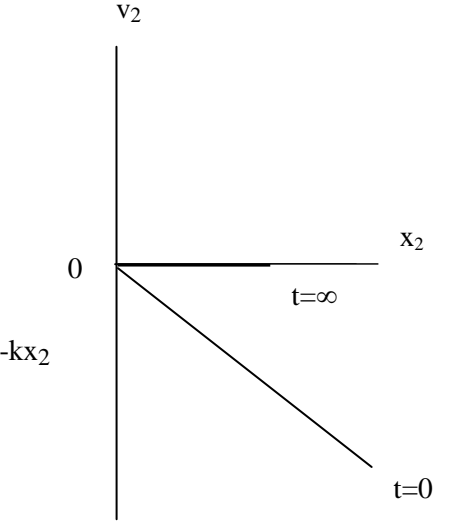
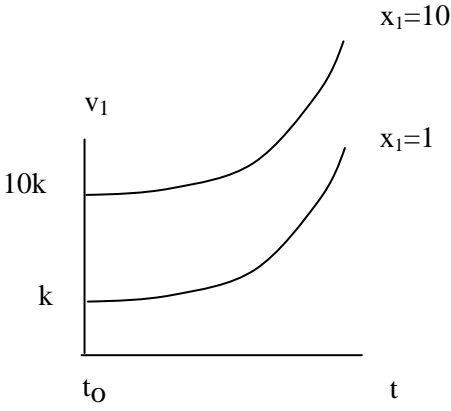
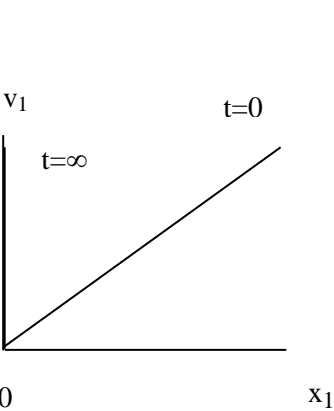
c) Plot the Eulerian and Lagrangian distributions

Lagrangian

$$v_1 = kx_1 \exp[k(t - t_0)]$$

$$v_2 = -kx_2 \exp[-k(t - t_0)]$$

$$v_3 = 0$$

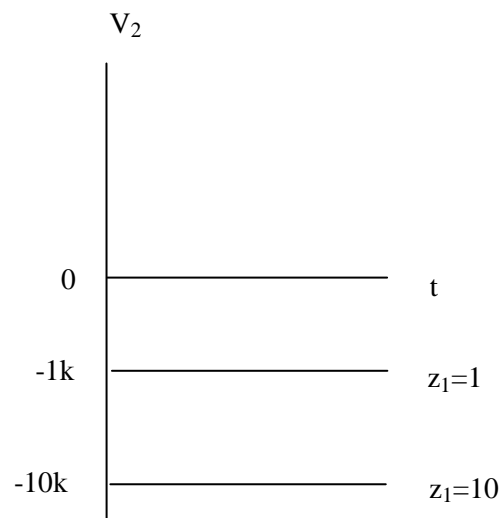
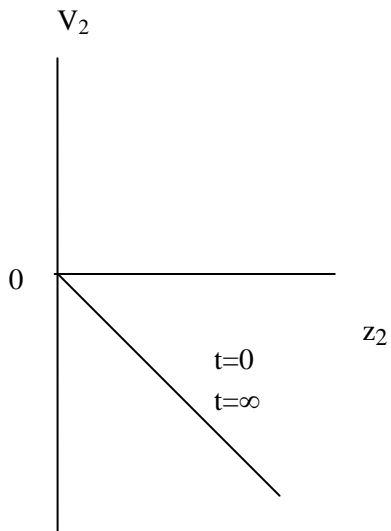
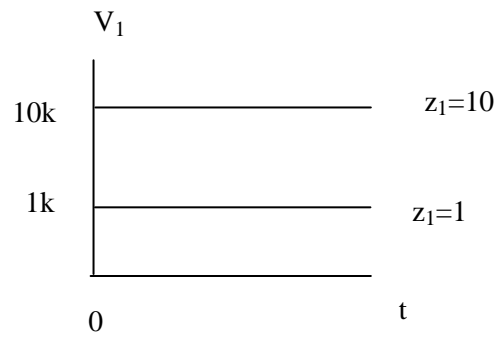
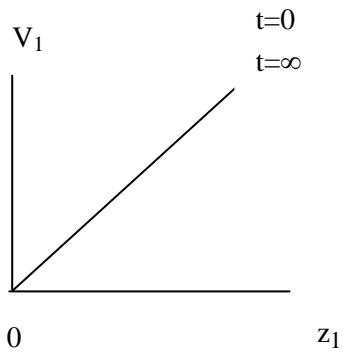


Eulerian

$$V_1 = k z_1$$

$$V_2 = -k z_2$$

$$V_3 = 0$$



$$V_1 = k z_1$$

$$V_2 = -k z_2$$

$$V_3 = 0$$

Equations for streamlines

$$\frac{dz_1}{d\lambda} = V_1 = k z_1 \qquad \frac{dz_2}{d\lambda} = V_2 = -k z_2$$

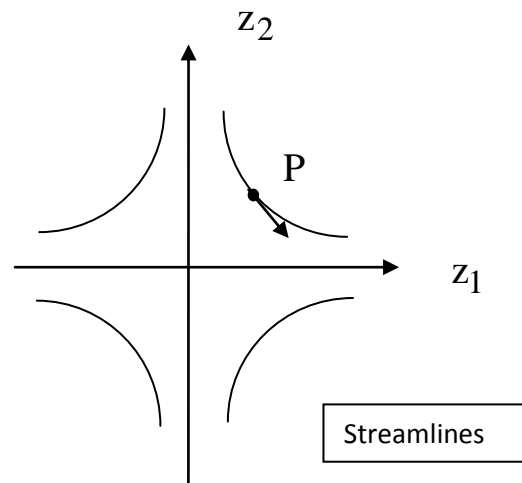
Pfaffian form

$$\frac{dz_1}{V_1} = \frac{dz_2}{V_2} = d\lambda$$

$$\frac{dz_1}{k z_1} = \frac{dz_2}{-k z_2}$$

$$\frac{dz_1}{z_1} + \frac{dz_2}{z_2} = 0$$

$$z_1 z_2 = C, \text{ a constant}$$



The tangent of the streamline is given by

$$\frac{dz_2}{dz_1} = \frac{C}{-z_2^2} = -\frac{z_2}{z_1}$$

The fluid flows along the tangent of the streamlines, and there can be no flow perpendicular to the streamlines.

3. Derive the Cayley-Hamilton equation using the Lagrange multiplier method and identify the principle strain invariants

γ_{ij}	strain components on a plane arbitrarily inclined from the principal planes
n_j	direction cosines that identify the inclined plane
γ_i	principle strains

The relationship between the principal strain components on the inclined plain is

$$\gamma_i = \gamma_{ij} n_j$$

Multiply by n_i

$$\gamma(n) = \gamma_i n_i = n_i n_j \gamma_{ij}$$

Let $\gamma = \lambda$ a scalar for one of the principal strain components

$$\gamma_i = \lambda n_i = \lambda \delta_{ij} n_j$$

By substitution,

$$\gamma_{ij} n_j = \lambda \delta_{ij} n_j$$

Multiply by n_i

$$\gamma_{ij} n_i n_j = \lambda \delta_{ij} n_i n_j = \lambda n_i n_i$$

Direction cosine property

$$\mathbf{n} \cdot \mathbf{n} = n_i n_i = 1$$

Now construct a scalar function f .

$$f = \gamma_{ij} n_i n_j - \lambda (n_i n_i - 1)$$

λ is the Lagrange multiplier

f is the work required for satisfying: $\gamma(\mathbf{n}) = \gamma_i n_i = n_i n_j \gamma_{ij}$

The goal is to find the direction of unit vector \mathbf{n} for which $\gamma(\mathbf{n})$ is maximum.

Take the derivative and set to zero.

$$\begin{aligned} \frac{\partial}{\partial n_k} (\gamma_{ij} n_i n_j) &= \gamma_{ij} \frac{\partial}{\partial n_k} (n_i n_j) = \gamma_{ij} \left[n_j \frac{\partial n_i}{\partial n_k} + n_i \frac{\partial n_j}{\partial n_k} \right] = \gamma_{ij} [n_j \delta_{ik} + n_i \delta_{jk}] \\ &= \gamma_{kj} n_j + \gamma_{ik} n_i = 2\gamma_{kj} n_j \quad (\text{by symmetry}) \end{aligned}$$

Also,

$$\frac{\partial}{\partial n_j} (\lambda n_i n_i) = \lambda \frac{\partial}{\partial n_j} (n_i n_i) = \lambda \left[n_i \frac{\partial n_i}{\partial n_j} + n_i \frac{\partial n_i}{\partial n_j} \right] = \lambda [n_i \delta_{ij} + n_i \delta_{ij}] = 2\lambda \delta_{ij} n_j$$

Thus

$$\frac{\partial}{\partial n_i} f = 2\gamma_{ij} n_j - 2\lambda \delta_{ij} n_j = 0$$

$$(\gamma_{ij} - \lambda \delta_{ij}) n_j = 0$$

Since n_j is arbitrary,

$$\det(\gamma_{ij} - \lambda \delta_{ij}) = 0$$

Cayley-Hamilton equation

$$\lambda^3 - I_1\lambda^2 + I_2\lambda - I_3 = 0$$

The standard eigenvalue problem is

$$(\gamma_{ij} - \lambda\delta_{ij})\mathbf{n}_j = 0$$

$$\det(\gamma_{ij} - \lambda\delta_{ij}) = 0$$

$$\begin{vmatrix} \gamma_{11} - \lambda & \gamma_{12} & \gamma_{13} \\ \gamma_{21} & \gamma_{22} - \lambda & \gamma_{23} \\ \gamma_{31} & \gamma_{32} & \gamma_{33} - \lambda \end{vmatrix} = 0$$

$$\begin{aligned} &+ (\gamma_{11} - \lambda)(\gamma_{22} - \lambda)(\gamma_{33} - \lambda) - (\gamma_{11} - \lambda)\gamma_{32}\gamma_{23} \\ &+ \gamma_{12}\gamma_{23}\gamma_{31} - \gamma_{12}\gamma_{21}(\gamma_{33} - \lambda) \\ &+ \gamma_{13}\gamma_{21}\gamma_{32} - \gamma_{31}\gamma_{13}(\gamma_{22} - \lambda) = 0 \end{aligned}$$

$$\begin{aligned} &+ (\gamma_{11} - \lambda)(\gamma_{22}\gamma_{33} - \lambda(\gamma_{22} + \gamma_{33}) + \lambda^2) \\ &- \gamma_{11}\gamma_{32}\gamma_{23} + \lambda\gamma_{32}\gamma_{23} \\ &+ \gamma_{12}\gamma_{23}\gamma_{31} - \gamma_{12}\gamma_{21}\gamma_{33} + \lambda\gamma_{12}\gamma_{21} \\ &+ \gamma_{13}\gamma_{21}\gamma_{32} - \gamma_{31}\gamma_{13}\gamma_{22} + \lambda\gamma_{31}\gamma_{13} = 0 \end{aligned}$$

$$\begin{aligned}
& -\lambda\gamma_{22}\gamma_{33} + \lambda^2(\gamma_{22} + \gamma_{33}) - \lambda^3 \\
& \gamma_{11}\gamma_{22}\gamma_{33} - \lambda(\gamma_{11}\gamma_{22} + \gamma_{11}\gamma_{33}) + \gamma_{11}\lambda^2 \\
& - \gamma_{11}\gamma_{32}\gamma_{23} + \lambda\gamma_{32}\gamma_{23} \\
& + \gamma_{12}\gamma_{23}\gamma_{31} - \gamma_{12}\gamma_{21}\gamma_{33} + \lambda\gamma_{12}\gamma_{21} \\
& + \gamma_{13}\gamma_{21}\gamma_{32} - \gamma_{31}\gamma_{13}\gamma_{22} + \lambda\gamma_{31}\gamma_{13} = 0
\end{aligned}$$

$$\begin{aligned}
& -\lambda^3 + (\gamma_{11} + \gamma_{22} + \gamma_{33})\lambda^2 - (\gamma_{11}\gamma_{22} - \gamma_{12}\gamma_{21} + \gamma_{11}\gamma_{33} - \gamma_{31}\gamma_{13} + \gamma_{22}\gamma_{33} - \gamma_{32}\gamma_{23})\lambda \\
& + \gamma_{11}\gamma_{22}\gamma_{33} - \gamma_{11}\gamma_{32}\gamma_{23} + \gamma_{12}\gamma_{23}\gamma_{31} - \gamma_{12}\gamma_{21}\gamma_{33} + \gamma_{13}\gamma_{21}\gamma_{32} - \gamma_{31}\gamma_{13}\gamma_{22} = 0
\end{aligned}$$

$$I_1 = \gamma_{11} + \gamma_{22} + \gamma_{33} = \gamma_{ii}$$

$$I_2 = \frac{1}{2}(\gamma_{ii}\gamma_{jj} - \gamma_{ij}\gamma_{ij})$$

$$I_3 = |\gamma_{ij}|$$

The eigenvalues λ are the principle strain invariants.

4. From the laboratory measurements the following strain tensor components are obtained.

$$\gamma_{ij} = \begin{bmatrix} 2 & 1 & 1 \\ 1 & 2 & 0 \\ 1 & 0 & 1 \end{bmatrix}$$

a) Determine the principle strains invariants

$$I_1 = \text{trace} \begin{bmatrix} 2 & 1 & 1 \\ 1 & 2 & 0 \\ 1 & 0 & 1 \end{bmatrix} = 5$$

$$I_2 = \begin{vmatrix} 2 & 0 \\ 0 & 1 \end{vmatrix} + \begin{vmatrix} 2 & 1 \\ 1 & 1 \end{vmatrix} + \begin{vmatrix} 2 & 1 \\ 1 & 2 \end{vmatrix} = 2 + 2 - 1 + 4 - 1 = 6$$

$$I_3 = \det \begin{bmatrix} 2 & 1 & 1 \\ 1 & 2 & 0 \\ 1 & 0 & 1 \end{bmatrix} = 4 - 1 - 2 = 1$$

b) Determine the principle strains

$$\lambda^3 - I_1\lambda^2 + I_2\lambda - I_3 = 0$$

Eigenvalues λ via Matlab

0.1981

1.555

3.247

The eigenvalues are the principles strains

c) Determine the principle direction matrix

Eigenvectors via Matlab

V =

-0.5910 0.3280 0.7370

0.3280 -0.7370 0.5910

0.7370 0.5910 0.3280

The eigenvectors represent the a_{ij} , which are the principal direction cosines

Verify your results by recalculating the principle strains from principal direction cosines.

Calculated using Matlab.

$$\mathbf{V}^T \boldsymbol{\gamma} \mathbf{V} = \begin{bmatrix} \lambda_1 & 0 & 0 \\ 0 & \lambda_2 & 0 \\ 0 & 0 & \lambda_3 \end{bmatrix} = \begin{bmatrix} 0.1981 & 0 & 0 \\ 0 & 1.555 & 0 \\ 0 & 0 & 3.247 \end{bmatrix}$$

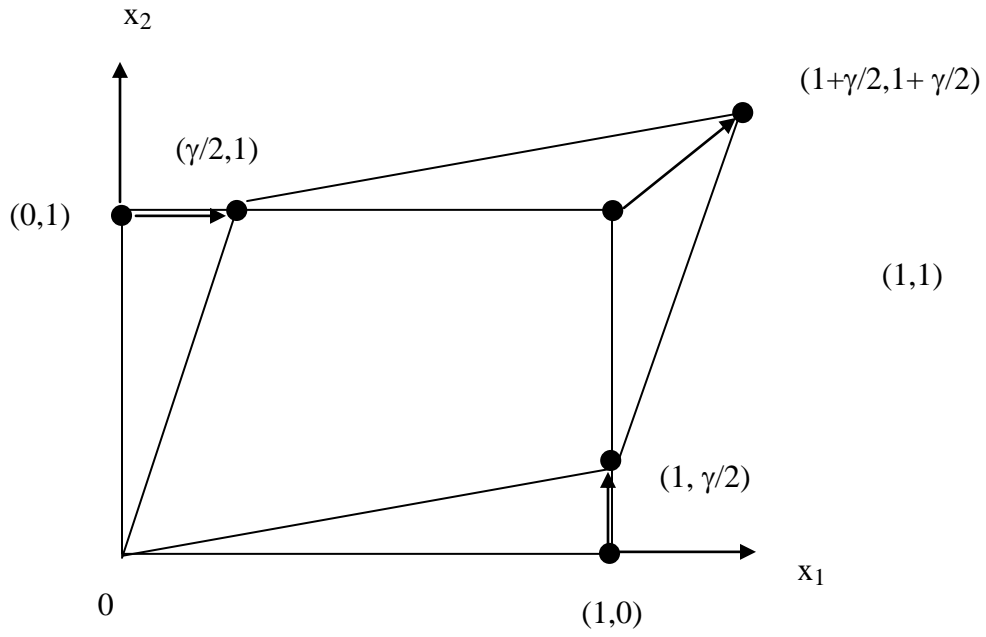
5. Sketch the displaced positions in the x_1x_2 - plane of the points initially on the sides of the square bounded by $x_1 = 0, x_1 = 1, x_2 = 0, x_2 = 1$.

(a) $\mathbf{u} = \frac{1}{2}\gamma x_2 \mathbf{i}_1 + \frac{1}{2}\gamma x_1 \mathbf{i}_2$

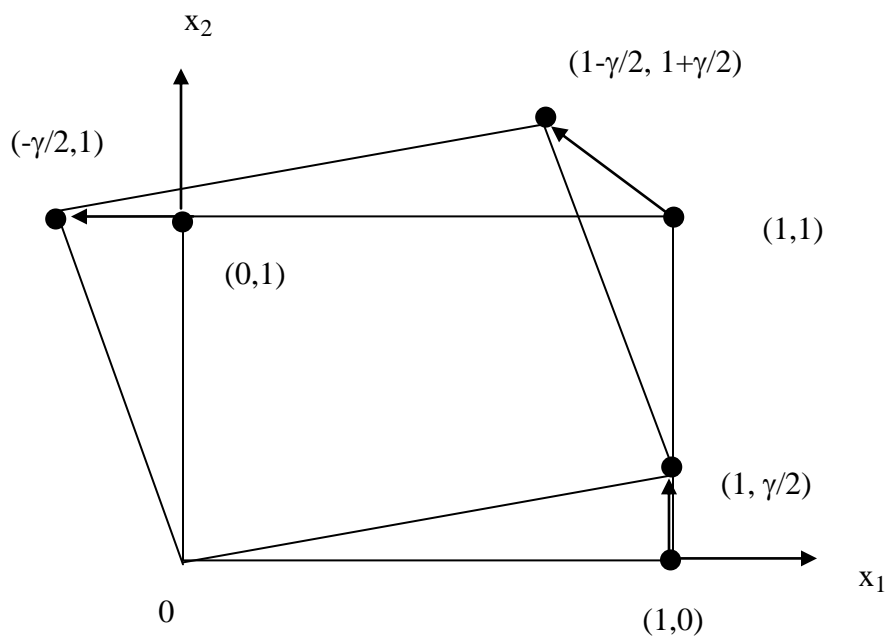
(b) $\mathbf{u} = -\frac{1}{2}\gamma x_2 \mathbf{i}_1 + \frac{1}{2}\gamma x_1 \mathbf{i}_2$

(c) $\mathbf{u} = \gamma x_1 \mathbf{i}_2$

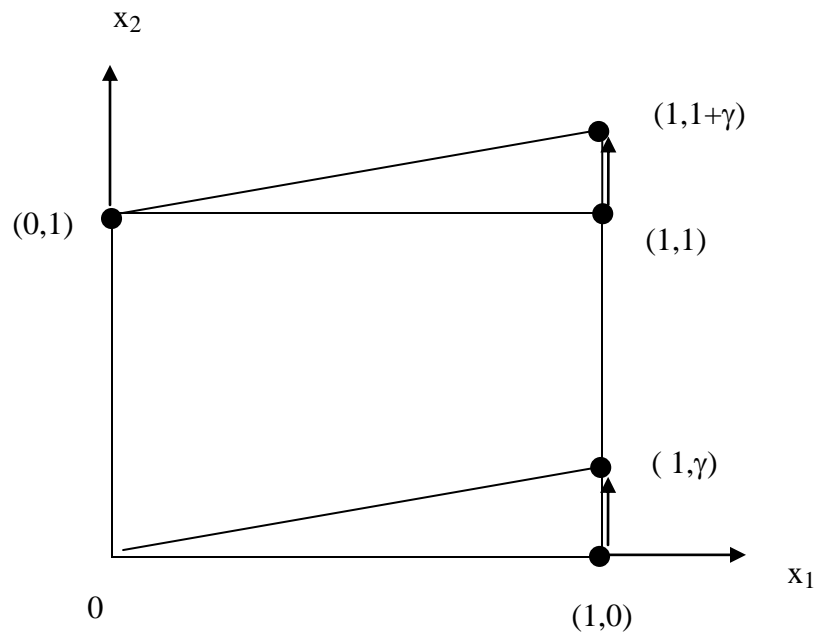
a)



b)



c)



6. For each of the displacement fields in P.5, determine the finite and infinitesimal strain tensors.

5. Sketch the displaced positions in the x_1x_2 -plane of the points initially on the sides of the square bounded by $x_1 = 0, x_1 = 1, x_2 = 0, x_2 = 1$.

(a) $\mathbf{u} = \frac{1}{2}\gamma x_2 \mathbf{i}_1 + \frac{1}{2}\gamma x_1 \mathbf{i}_2$

(b) $\mathbf{u} = -\frac{1}{2}\gamma x_2 \mathbf{i}_1 + \frac{1}{2}\gamma x_1 \mathbf{i}_2$

(c) $\mathbf{u} = \gamma x_1 \mathbf{i}_2$

a)

$$\mathbf{u} = \frac{1}{2}\gamma x_2 \mathbf{i}_1 + \frac{1}{2}\gamma x_1 \mathbf{i}_2$$

$$\nabla \mathbf{u} = \begin{bmatrix} \frac{\partial \phi_1}{\partial x_1} & \frac{\partial \phi_1}{\partial x_2} \\ \frac{\partial \phi_2}{\partial x_1} & \frac{\partial \phi_2}{\partial x_2} \end{bmatrix} = \begin{bmatrix} 0 & \frac{1}{2}\gamma \\ \frac{1}{2}\gamma & 0 \end{bmatrix}$$

$$\mathbf{E}_{\text{linear}} = \frac{1}{2}(\nabla \mathbf{u} + \nabla \mathbf{u}^T) = \frac{1}{2} \left\{ \begin{bmatrix} 0 & \frac{1}{2}\gamma \\ \frac{1}{2}\gamma & 0 \end{bmatrix} + \begin{bmatrix} 0 & \frac{1}{2}\gamma \\ \frac{1}{2}\gamma & 0 \end{bmatrix} \right\} = \begin{bmatrix} 0 & \frac{1}{2}\gamma \\ \frac{1}{2}\gamma & 0 \end{bmatrix}$$

$$\mathbf{E}_{\text{finite}} = \frac{1}{2}(\nabla \mathbf{u} + \nabla \mathbf{u}^T + \nabla \mathbf{u}^T \nabla \mathbf{u}) = \begin{bmatrix} 0 & \frac{1}{2}\gamma \\ \frac{1}{2}\gamma & 0 \end{bmatrix} + \frac{1}{2} \begin{bmatrix} 0 & \frac{1}{2}\gamma \\ \frac{1}{2}\gamma & 0 \end{bmatrix} \begin{bmatrix} 0 & \frac{1}{2}\gamma \\ \frac{1}{2}\gamma & 0 \end{bmatrix}$$

$$\begin{aligned}
\mathbf{E}_{\text{finite}} &= \frac{1}{2}(\nabla\mathbf{u} + \nabla\mathbf{u}^T + \nabla\mathbf{u}^T\nabla\mathbf{u}) = \begin{bmatrix} 0 & \frac{1}{2}\gamma \\ \frac{1}{2}\gamma & 0 \end{bmatrix} + \frac{1}{2} \begin{bmatrix} 0 & \frac{1}{2}\gamma \\ \frac{1}{2}\gamma & 0 \end{bmatrix} \begin{bmatrix} 0 & \frac{1}{2}\gamma \\ \frac{1}{2}\gamma & 0 \end{bmatrix} \\
&= \begin{bmatrix} \frac{1}{8}\gamma^2 & \frac{1}{2}\gamma \\ \frac{1}{2}\gamma & \frac{1}{8}\gamma^2 \end{bmatrix} + \frac{1}{2} \begin{bmatrix} \frac{1}{4}\gamma^2 & 0 \\ 0 & \frac{1}{4}\gamma^2 \end{bmatrix} \\
&= \begin{bmatrix} 0 & \frac{1}{2}\gamma \\ \frac{1}{2}\gamma & 0 \end{bmatrix} + \begin{bmatrix} \frac{1}{8}\gamma^2 & 0 \\ 0 & \frac{1}{8}\gamma^2 \end{bmatrix} = \begin{bmatrix} \frac{1}{8}\gamma^2 & \frac{1}{2}\gamma \\ \frac{1}{2}\gamma & \frac{1}{8}\gamma^2 \end{bmatrix}
\end{aligned}$$

b)

$$\mathbf{u} = -\frac{1}{2}\gamma x_2 \mathbf{i}_1 + \frac{1}{2}\gamma x_1 \mathbf{i}_2$$

$$\nabla\mathbf{u} = \begin{bmatrix} \frac{\partial\phi_1}{\partial x_1} & \frac{\partial\phi_1}{\partial x_2} \\ \frac{\partial\phi_2}{\partial x_1} & \frac{\partial\phi_2}{\partial x_2} \end{bmatrix} = \begin{bmatrix} 0 & \frac{1}{2}\gamma \\ -\frac{1}{2}\gamma & 0 \end{bmatrix}$$

$$\mathbf{E}_{\text{linear}} = \frac{1}{2}(\nabla\mathbf{u} + \nabla\mathbf{u}^T) = \frac{1}{2} \left\{ \begin{bmatrix} 0 & \frac{1}{2}\gamma \\ -\frac{1}{2}\gamma & 0 \end{bmatrix} + \begin{bmatrix} 0 & -\frac{1}{2}\gamma \\ \frac{1}{2}\gamma & 0 \end{bmatrix} \right\} = \begin{bmatrix} 0 & 0 \\ 0 & 0 \end{bmatrix}$$

$$\begin{aligned} \mathbf{E}_{\text{finite}} &= \frac{1}{2}(\nabla\mathbf{u} + \nabla\mathbf{u}^T + \nabla\mathbf{u}^T\nabla\mathbf{u}) = \begin{bmatrix} 0 & 0 \\ 0 & 0 \end{bmatrix} + \frac{1}{2} \begin{bmatrix} 0 & -\frac{1}{2}\gamma \\ \frac{1}{2}\gamma & 0 \end{bmatrix} \begin{bmatrix} 0 & \frac{1}{2}\gamma \\ -\frac{1}{2}\gamma & 0 \end{bmatrix} \\ &= \begin{bmatrix} \frac{1}{8}\gamma^2 & 0 \\ 0 & \frac{1}{8}\gamma^2 \end{bmatrix} \end{aligned}$$

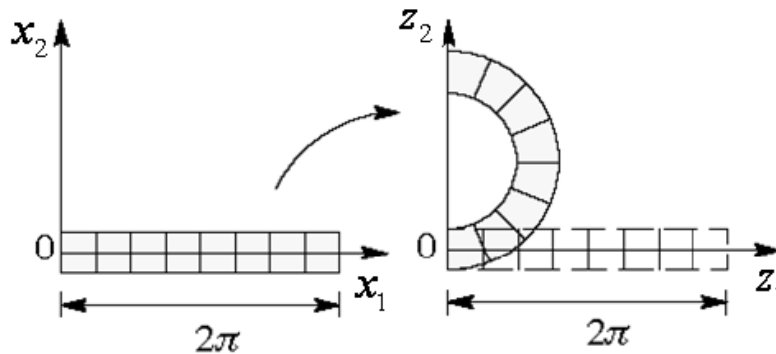
C) $\mathbf{u} = \gamma x_1 \mathbf{i}_2$

$$\nabla\mathbf{u} = \begin{bmatrix} \frac{\partial\phi_1}{\partial x_1} & \frac{\partial\phi_1}{\partial x_2} \\ \frac{\partial\phi_2}{\partial x_1} & \frac{\partial\phi_2}{\partial x_2} \end{bmatrix} = \begin{bmatrix} 0 & 0 \\ \gamma & 0 \end{bmatrix}$$

$$\mathbf{E}_{\text{linear}} = \frac{1}{2}(\nabla\mathbf{u} + \nabla\mathbf{u}^T) = \frac{1}{2} \begin{bmatrix} 0 & \gamma \\ \gamma & 0 \end{bmatrix}$$

$$\begin{aligned} \mathbf{E}_{\text{finite}} &= \frac{1}{2}(\nabla\mathbf{u} + \nabla\mathbf{u}^T + \nabla\mathbf{u}^T\nabla\mathbf{u}) = \frac{1}{2} \begin{bmatrix} 0 & \gamma \\ \gamma & 0 \end{bmatrix} + \frac{1}{2} \begin{bmatrix} 0 & \gamma \\ \gamma & 0 \end{bmatrix} \begin{bmatrix} 0 & \gamma \\ \gamma & 0 \end{bmatrix} \\ &= \frac{1}{2} \begin{bmatrix} 0 & \gamma \\ \gamma & 0 \end{bmatrix} + \frac{1}{2} \begin{bmatrix} \gamma^2 & 0 \\ 0 & \gamma^2 \end{bmatrix} = \frac{1}{2} \begin{bmatrix} \gamma^2 & \gamma \\ \gamma & \gamma^2 \end{bmatrix} \end{aligned}$$

7. (Based on Hjelmstad's book) Consider the large deformation of a flexible beam (strip) under bending that deforms the strip of length 2π (say inches) into a semicircular arc, as shown below. Assume the beam is inextensible; that is, the length of the middle line (given by $x_2 = 0$), 2π , remains unchanged after the bending (the top surface and bottom surface respectively becomes shorter and longer than 2π , as shown in the figure). Let the depth (in the x_2 direction) be $2c$. For a narrow strip ($c \ll \pi$), the change in the depth can be neglected.



a. Construct the deformation map, that is, find the mathematical expression

$$\mathbf{R} = \mathbf{R}(\mathbf{r}, t), \text{ or, } z_i = z_i(x_j, t)$$

What is the location of the point originally located at $\mathbf{r} = 2\pi\mathbf{i}_1$?

b. Compute the deformation gradient tensor \mathbf{F} ;

c. Determine the Lagrangian (finite) strain tensor.

diameter = 4

radius=2

$$\underline{\phi}(\mathbf{x}) = [(2 - x_2)\sin(x_1/2)]\mathbf{i}_1 + [2 - (2 - x_2)\cos(x_1/2)]\mathbf{i}_2 + x_3\mathbf{i}_2$$

The point at $\mathbf{r} = 2\pi\mathbf{i}_1$ maps to $(0, 4, 0)$ in the new coordinate system.

Or $4\mathbf{i}_2$

b)

$$\mathbf{F} = \nabla \underline{\phi} = \begin{bmatrix} \frac{\partial \phi_1}{\partial x_1} & \frac{\partial \phi_1}{\partial x_2} & \frac{\partial \phi_1}{\partial x_3} \\ \frac{\partial \phi_2}{\partial x_1} & \frac{\partial \phi_2}{\partial x_2} & \frac{\partial \phi_2}{\partial x_3} \\ \frac{\partial \phi_3}{\partial x_1} & \frac{\partial \phi_3}{\partial x_2} & \frac{\partial \phi_3}{\partial x_3} \end{bmatrix} = \begin{bmatrix} \frac{1}{2}(2-x_2)\cos(x_1/2) & -\sin(x_1/2) & 0 \\ -\frac{1}{2}(2-x_2)\sin(x_1/2) & \cos(x_1/2) & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

c)

The Green deformation tensor is:

$$\mathbf{C} = \mathbf{F}^T \mathbf{F}$$

$$= \begin{bmatrix} \frac{1}{2}(2-x_2)\cos(x_1/2) & -\frac{1}{2}(2-x_2)\sin(x_1/2) & 0 \\ -\sin(x_1/2) & \cos(x_1/2) & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} \frac{1}{2}(2-x_2)\cos(x_1/2) & -\sin(x_1/2) & 0 \\ -\frac{1}{2}(2-x_2)\sin(x_1/2) & \cos(x_1/2) & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

$$= \begin{bmatrix} \frac{1}{4}(2-x_2)^2 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} = \begin{bmatrix} \left(1 - \frac{1}{2}x_2\right)^2 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

The Lagrangian strain tensor \mathbf{E} is defined to be have the difference between the Green deformation tensor and the identity tensor \mathbf{I} as

$$\mathbf{E} \equiv \frac{1}{2}[\mathbf{C} - \mathbf{I}] = \frac{1}{2} \left\{ \begin{bmatrix} \frac{1}{4}(2-x_2)^2 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} - \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \right\}$$

$$= \frac{1}{2} \left\{ \begin{bmatrix} \frac{1}{4}(2-x_2)^2 - 1 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} \right\}$$

$$= \frac{1}{2} \begin{bmatrix} \frac{1}{4}(4-4x_2+x_2^2) - 1 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

$$= \frac{1}{2} \begin{bmatrix} \left(1-x_2 + \frac{1}{4}x_2^2\right) - 1 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

$$= \frac{1}{2} \begin{bmatrix} -x_2 + \frac{1}{4}x_2^2 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

$$= \begin{bmatrix} -\frac{1}{2}x_2 + \frac{1}{8}x_2^2 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$