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Rotating unbalance in machinery is an example of an applied force. Model a machine with rotating unbalance as a single-degree-of-freedom system.



Figure 1.

The variables are:

- M is the total mass
- m is the eccentric mass representing the unbalance
- c is the viscous damping coefficient
- k is the stiffness
- x is the absolute displacement of the non-rotating mass
- e is the eccentricity
- $\omega$  is the rotational frequency in radians/sec

The free-body diagram is





The rotating unbalance force is

$$f(t) = -m \frac{d^2}{dt^2} \left[ e \sin \omega t \right]$$
(1)

$$f(t) = m\omega^2 e \sin \omega t$$
 (2)

Sum the forces using Newton's law. The upward direction is positive.

$$\sum \mathbf{F} = \mathbf{m}\ddot{\mathbf{x}} \tag{3}$$

$$M\ddot{x} = f(t) - kx - c\dot{x}$$
<sup>(4)</sup>

$$M\ddot{x} + c\dot{x} + kx = f(t)$$
(5)

$$M\ddot{x} + c\dot{x} + kx = m\omega^2 e \sin\omega t$$
(6)

$$\ddot{\mathbf{x}} + (\mathbf{c}/\mathbf{M})\dot{\mathbf{x}} + (\mathbf{k}/\mathbf{M})\mathbf{x} = (\mathbf{m}/\mathbf{M})\omega^2 \,\mathbf{e}\,\sin\omega\mathbf{t} \tag{7}$$

$$c/M = 2\xi\omega_n \tag{8}$$

$$\omega_{\rm n} = \sqrt{\frac{\rm k}{\rm m}} \tag{9}$$

$$\ddot{\mathbf{x}} + 2\xi \omega_n \dot{\mathbf{x}} + \omega_n^2 \mathbf{x} = (m/M)\omega^2 \mathbf{e} \sin \omega t$$
(10)

Equation (10) can be solved via Reference 1.

The steady-state displacement magnitude X is

$$X = \left\{ \frac{m\omega^2 e}{k} \right\} \frac{1}{\sqrt{\left(1 - \rho^2\right)^2 + \left(2\xi\rho\right)^2}}$$
(11)

where

$$\rho = \frac{\omega}{\omega_n}$$

The steady-state displacement magnitude can be scaled as

$$\frac{M}{me}X = \left\{\frac{M\omega^2}{k}\right\} \frac{1}{\sqrt{\left(1-\rho^2\right)^2 + \left(2\xi\rho\right)^2}}$$
(12)

$$\frac{M}{me}X = \left\{\frac{\omega^2}{\omega_n^2}\right\} \frac{1}{\sqrt{\left(1-\rho^2\right)^2 + \left(2\xi\rho\right)^2}}$$
(13)

$$\frac{M}{me}X = \frac{\rho^2}{\sqrt{\left(1 - \rho^2\right)^2 + \left(2\xi\rho\right)^2}}$$
(14)

The corresponding phase angle is

$$\theta = \arctan\left[\frac{2\xi\rho}{1-\rho^2}\right]$$
(15)

## <u>Reference</u>

1. T. Irvine, The Time-Domain Response of a Single-Degree-of-Freedom System Subjected to a Sinusoidal Force, Vibrationdata, 1999.