MODAL GAIN FACTORS IN LAUNCH VEHICLE AUTOPILOT STABILITY ANALYSIS

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Variables

[I]	=	Identity matrix
ξ	=	Modal damping ratio
ω	=	Natural frequency (rad/sec)
Ω	=	Forcing frequency (rad/sec)
q	=	Modal displacement
Х	=	Physical displacement
[ø]	=	Mass normalized eigenvector matrix
F	=	Applied force
Н	=	Mobility transfer function
t	=	Time

Equation of Motion

Consider a launch vehicle represented in terms of a linearized structural dynamics equation. The equation includes bending modes primarily, but it may also contain slosh modes if the booster has liquid propellant.

The equation in decoupled modal coordinates is

$$[I]{\ddot{q}} + [2\xi\omega]{\dot{q}} + [\omega^2]{q} = [\phi]^T F$$
⁽¹⁾

Assume a harmonic force.

$$\mathbf{F} = \overline{\mathbf{F}}(\Omega) \exp(\mathbf{j}\,\Omega\,\mathbf{t}) \tag{2}$$

The modal displacement is represented as.

$$q = \overline{q}(\Omega) \exp(j\Omega t)$$
(3)

The modal velocity is

$$\dot{\mathbf{q}} = \mathbf{j} \Omega \overline{\mathbf{q}}(\Omega) \exp(\mathbf{j} \Omega \mathbf{t})$$
 (4)

The modal acceleration is

$$\ddot{q} = -\Omega^2 \ \overline{q}(\Omega) \exp(j\Omega t)$$
 (5)

By substitution,

$$\left[-\Omega^{2} + j2\xi\omega\Omega + \omega^{2}\right]\left\{\overline{q}(\Omega)\exp(j\Omega t)\right\} = \left[\phi\right]^{T}\overline{F}(\Omega)\exp(j\Omega t)$$
(6)

$$\left\{\omega^{2} - \Omega^{2} + j2\xi\omega\Omega\right\}\overline{q}(\Omega) = \left[\phi\right]^{T}\overline{F}(\Omega)$$
(7)

$$\overline{q}(\Omega) = \frac{\left[\phi\right]^{T} \overline{F}(\Omega)}{\left\{\omega^{2} - \Omega^{2} + j2\xi\omega\Omega\right\}}$$
(8)

The largest displacement response occurs at resonance where $\ \omega=\Omega$.

$$\overline{q}_{max} = \frac{\left[\phi\right]^{T} \overline{F}}{j2\xi\omega^{2}}$$
(9)

Note that

$$\dot{\overline{q}}(\Omega) = j \Omega \overline{q}(\Omega) \tag{10}$$

At resonance,

$$\dot{\overline{q}}(\omega) = j \,\omega \overline{q}(\omega) \tag{11}$$

The corresponding velocity at resonance is

$$\dot{q}_{max} = \frac{\left[\phi\right]^{T}\overline{F}}{2\xi\omega}$$
(12)

The physical velocity is related to the modal velocity by

$$\{\dot{\mathbf{x}}\} = \left[\phi\right]\!\!\left\{\dot{\mathbf{q}}\right\} \tag{13}$$

By substitution,

$$\{\dot{\mathbf{x}}\} = \frac{\left[\phi\right]\left[\phi\right]^{\mathrm{T}}\overline{\mathrm{F}}}{2\xi\omega}$$
(14)

The modal gain factor matrix for a given mode r driven at resonance is

$$\frac{\phi_{\rm r} \phi_{\rm r}^{\rm T}}{2\xi_{\rm r} \omega_{\rm r}} \tag{15}$$

Note that the gain factor varies with each pair of physical degrees-of-freedom.

The variation may be spatial. The variation may also be between a translational and rotational physical degree-of-freedom.

Furthermore, modes with higher gain factors tend to be more significant for control system design. Slosh modes are an exception, however.

Information for a specific application is given in Reference 1. The approach in Reference 1 shows the type of degrees-of-freedom to be included in the eigenvectors for a practical example.

The corresponding magnification factor for a given mode is

$$\frac{1}{2\xi} \tag{16}$$

<u>Reference</u>

1. T. Irvine, Notes on Mode Shapes and Bending Gain Units in Nastran, Vibrationdata, 2003.

APPENDIX A

Mobility Transfer Function

$$\overline{q}(\Omega) = \frac{\left[\phi\right]^{T} \overline{F}(\Omega)}{\left\{\omega^{2} - \Omega^{2} + j2\xi\omega\Omega\right\}}$$
(A-1)

$$\dot{\bar{q}}(\Omega) = \frac{j \Omega \left[\phi\right]^{T} \overline{F}(\Omega)}{\left\{\omega^{2} - \Omega^{2} + j2\xi\omega\Omega\right\}}$$
(A-2)

Furthermore,

$$\left\{ \dot{\overline{x}} \right\} = \left[\phi \right] \left\{ \dot{\overline{q}} \right\} = \frac{j \Omega \left[\phi \right] \left[\phi \right]^{T} \overline{F}(\Omega)}{\omega^{2} - \Omega^{2} + j2\xi\omega\Omega}$$
(A-3)

The mobility transfer function (velocity/force) for between any pair physical locations is

$$H_{ij}(\Omega) = \sum_{r=1}^{n} \left\{ \frac{j \Omega \phi_{ir} \phi_{jr}}{\omega_r^2 - \Omega^2 + j2\xi_r \omega_r \Omega} \right\}$$
(A-4)

The subscript r is mode index.

The number of modes is n.

The subscripts i and j refer to physical locations. Note that there is reciprocity. So i is the response location if j is the force input location, and vice-versa.

The mobility transfer function simplifies to

$$H_{ij}(\Omega) = j \Omega \sum_{r=1}^{n} \left\{ \frac{\phi_{ir} \phi_{jr}}{\omega_r^2 - \Omega^2 + j2\xi_r \omega_r \Omega} \right\}$$
(A-5)